

Task space tracking for manipulators

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For the purpose of controlling a manipulator in the task space, a linear model with task space position and velocity as state variables can be developed. This is done by means of exact compensation of the state space model non-linearities using non-linear feedback. In this paper, feedback control for this linear state space model is developed using optimal control theory. Integral action is included to compensate for unmodeled forces and torques. In the resulting control system, the problem of transforming the task space trajectory to the joint space is avoided, and the controller parameters can be chosen to satisfy requirements specified in the task space. Simulation experiments show promising results.

1. Introduction

In most applications for industrial manipulators, the desired trajectory or path is specified in the task space, e.g. a Cartesian co-ordinate system.

For task space trajectory tracking, the trajectory is normally transformed from task space to joint space. The tracking problem is then formulated in joint space. This can be done by using one PID-controller in each of the servomechanisms (Luh 1983). In certain applications this might be adequate, but as the controller parameters must be chosen to ensure stability in the worst case, the controllers will not be well tuned for all of the joint space. Alternatively, the non-linear terms of the dynamic equations may be compensated for by non-linear feedback. The resulting linear system can then be controlled using linear theory, e.g. optimal control theory (Luo and Saridis 1985) or the inverse problem technique (Paul 1972).

The transformation of the trajectory from task space to joint space, which is complex and multi-valued, can be avoided by formulating the control problem in the task space. This is done by Luh, Walker and Paul (1980a) in their resolved acceleration control scheme and by Tarn, Bejczy, Isidori and Chen (1984) who extend the ideas of Freund (1982) and employ a diffeomorphic state transformation which gives external linearization and output decoupling. In this way, a linear state space model with task space position and velocity as state variables is obtained. This linear system is controlled using a sub-optimal controller.

In this paper, the model structure proposed by Tarn *et al.* (1984) is derived in a straightforward manner in § 2 without the use of complex transformations. In § 3 a feedback controller is designed using optimal control theory. The use of integral action to compensate for unmodeled forces and torques is discussed. In § 4, the controller performance is demonstrated by means of simulation experiments.

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2. The state space model

The equations of motion for a general manipulator can be found from Newton-Euler's equation (Symon 1, Luh, Walker and Paul 1980b). We consider a manipulator with n joints. We have:

$$M(q)\ddot{q} + V\dot{q} + n(q, \dot{q}) + g(q) = \tau \quad (1)$$

where

- q vector representing the actual displacements of the n joints
- $M(q)$ inertia matrix
- V viscous friction matrix
- $n(q, \dot{q})$ vector defining Coriolis and centrifugal terms
- $g(q)$ vector defining the gravity terms
- τ vector of input generalized forces

The relationship between the velocity, \dot{p} , in the task space and the velocity, \dot{q} , in joint space is given by (Whitney 1972)

$$\dot{p} = J(q)\dot{q} \quad (2)$$

where $J(q)$ is the Jacobian matrix defined by $J_{ij} = \partial p_i / \partial q_j$.

For task space trajectory tracking, we choose the state vector $x = [x_1^T, x_2^T]^T$ where $x_1 = p$ which is the position in the task space and $x_2 = \dot{p}$ which is the velocity in the task space.

By differentiation of (2) with respect to time and combining the result with (1), we get the state space model

$$\dot{x}_1 = x_2 \quad (3)$$

$$\dot{x}_2 = \dot{J}(q)\dot{q} + J(q)M(q)^{-1}[-V\dot{q} - n(q, \dot{q}) - g(q) + \tau] \quad (4)$$

where $\dot{J}(q) = [\partial J(q) / \partial q]^T \dot{q}$.

By choosing

$$\tau = V\dot{q} + n(q, \dot{q}) + g(q) - M(q)J(q)^{-1}\dot{J}(q)\dot{q} + \Delta\tau$$

we obtain exact compensation of the non-linear terms in (4), and we get the state space model proposed by Tarn *et al.* (1984).

$$\dot{x}_1 = x_2 \quad (5)$$

$$\dot{x}_2 = u \quad (6)$$

where $u = J(q)M(q)^{-1}\Delta\tau$. The input generalized forces are found from

$$\tau = V\dot{q} + n(q, \dot{q}) + g(q) + M(q)J^{-1}(q)(-\dot{J}(q)\dot{q} + u) \quad (7)$$

We see from (7) that the Jacobian matrix $J(q)$ has to be non-singular. This means that the method of obtaining a linear model (5, 6) by exact compensation of the non-linear terms in (4) is not immediately applicable to kinematically redundant manipulators.

The method of obtaining a simple linear state space model by exact compensation of system non-linearities has previously been applied, e.g. to dynamic positioning of surface vessels (Balchen, Jenssen, Mathiesen and Saeldid, 1980).

3. Feedback control

We see that the system (5, 6) consists of n decoupled double integrators

$$\dot{x}_i = x_{i+n} \quad (8)$$

$$\dot{x}_{i+n} = u_i \quad (9)$$

where $1 \leq i \leq n$. As a result of this, the problem of finding the suitable feedback control for the system (5, 6) is reduced to the n identical problems of finding a suitable control for the systems (8, 9).

In this paper, optimal control theory is used in feedback controller design. However, classical servomechanism theory may also be used.

For trajectory tracking in the case where close tracking is required and where moderate overshoot is tolerated, we want to minimize the functional

$$J_i = \int_{t_0}^{t_f} [q_{ii}(\Delta x_i)^2 + q_{i+n, i+n}(\Delta x_{i+n})^2 + p_{ii} u_i^2] dt \quad (10)$$

where t_0 to t_f is the time interval of the tracking, $\Delta x_i = x_i - x_{i0}$ and $\Delta x_{i+n} = x_{i+n} - x_{i+n,0}$ where x_{i0} and $x_{i+n,0}$ are the position and velocity setpoints. q_{ii} , $q_{i+n, i+n}$ and p_{ii} are chosen as $q_{ii} = (\Delta x_{i, \max})^{-2}$, $q_{i+n, i+n} = (\Delta x_{i+n, \max})^{-2}$ and $p_{ii} = (u_{i, \max})^{-2}$ where $\Delta x_{i, \max}$, $\Delta x_{i+n, \max}$ and $u_{i, \max}$ are the maximum acceptable values for the state deviation and for the control.

For simplicity, we let $t_f \rightarrow \infty$. The resulting feedback is given by (Athans and Falb 1966)

$$u_i = g_i \Delta x_i + g_{i+n} \Delta x_{i+n} \quad (11)$$

where $g_i = -\sqrt{(q_{ii}/p_{ii})}$ and $g_{i+n} = -\sqrt{[2\sqrt{(q_{ii}/p_{ii})} + q_{i+n, i+n}/p_{ii}]}$. To compensate for constant disturbances, we include integral action by choosing

$$u_i = g_i \Delta x_i + g_{i+n} \Delta x_{i+n} + K_i \int_{t_0}^t \Delta x_i(\tau) d\tau \quad (12)$$

An appropriate choice of K_i is $K_i = g_i^2/(5g_{i+n})$ which corresponds to choosing $T_i = 5T_D$ in a PID-controller where T_i is the integral time and T_D is the derivative time. If the desired acceleration \ddot{x}_{i0} is available, feedforward should be implemented giving the control

$$u_i = \ddot{x}_{i0} + g_i \Delta x_i + g_{i+n} \Delta x_{i+n} + K_i \int_{t_0}^t \Delta x_i(\tau) d\tau \quad (13)$$

The feedback given by (13) has a simple structure, and the controller parameters are found from the specified requirements in the task space given in terms of acceptable state deviation and the maximum acceptable task space acceleration. Of course these specifications may have to be adjusted so that the bandwidth is lower than the resonant frequency of the manipulator.

If overshoot is not allowable, g_{i+n} should be adjusted so that the relative damping ζ is greater than unity. This can also be done by choosing a small $x_{i+n, \max}$. The resulting control system is shown in Fig. 1.

In the case where x_1 is the position in Cartesian space and orientation in terms of Euler angles, the control given by (13) without the integral term and by (7) is of the same form as resolved acceleration control (Luh *et al.* 1980a). In this method, (7) is computed using the recursive Newton-Euler formulation (Luh *et al.* 1980b).

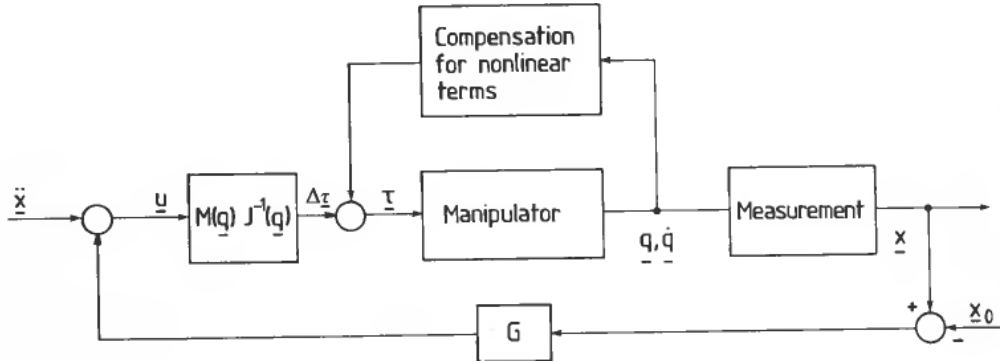


Figure 1. Control system.

In Tarn and co-workers (1984), the double integrators (8, 9) are stabilized by feedback $u_i = f_i x_i + f_{i+n} x_{i+n}$. Then a linear quadratic criterion is minimized for the resulting damped harmonic oscillator. Clearly this gives a sub-optimal solution, as the optimal control should be applied to the original double integrators. It is also reported by Tarn *et al.* (1984) that choice of f_i and f_{i+n} has little influence on system behaviour.

The controller design in this section is based upon the assumption that the model (3, 4) is perfectly known. In practice this will not be the case due to modeling errors or model simplifications. The question of the robustness of the controller is therefore raised. This problem is investigated by Spong and Vidyasagar (1985) who present an explicit bound on the control deviation as a function of model uncertainty. A crucial point in their derivation, is the assumption that the difference between the actual and computed value of the inertia matrix M is small.

4. Simulation

The control system developed in sections 2 and 3 was tested for the positioning part of an industrial manipulator with rotary joints. This was done by means of simulation experiments.

The manipulator is shown in Fig. 2. Only the inner three joints are considered. The equations of motion are (Saridis, 1983)

$$M(q)\dot{\omega} + n(q, \omega) + q(q) = \sigma$$

where

$$M(q) = \begin{bmatrix} J_{11} + J_{22}(C2)^2 + J_{33}[C(2+3)]^2 + J_{23}C2C[2+3] & 0 & 0 \\ 0 & J_{22} & J_{23}C3 \\ 0 & J_{23}C3 & J_{33} \end{bmatrix},$$

$$n(q, \omega) = - \begin{bmatrix} [2J_{23}S2C(2+3) + 2J_{22}S2C2]\omega_1\omega_2 \\ \quad + [2J_{23}S(2+3)C2 + 2J_{33}S(2+3)C(2+3)]\omega_1\omega_3 \\ - [J_{23}C(2+3)S2 + J_{22}C2S2]\omega_1^2 + J_{23}S3\omega_3^2 \\ \quad + M_2S2 + M_3S(2+3) \\ - [J_{23}C2S(2+3) + J_{33}C(2+3)S(2+3)]\omega_1^2 \\ \quad - J_{23}S3\omega_2^2 + M_3S(2+3) \end{bmatrix},$$

$$\omega = \begin{bmatrix} \omega_1 \\ \omega_2 \\ \omega_3 \end{bmatrix} = \begin{bmatrix} \dot{q}_1 \\ \dot{q}_2 \\ \dot{q}_2 + \dot{q}_3 \end{bmatrix},$$

$\sigma = [\tau_1, \tau_2 - \tau_3, \tau_3]^T$ where τ_i is the torque applied at joint i ,
 $C2 = \cos q_2$, $C(2+3) = \cos(q_2 + q_3)$, $C3 = \cos q_3$,
 $S2 = \sin q_2$, $S(2+3) = \sin(q_2 + q_3)$, $S3 = \sin q_3$,

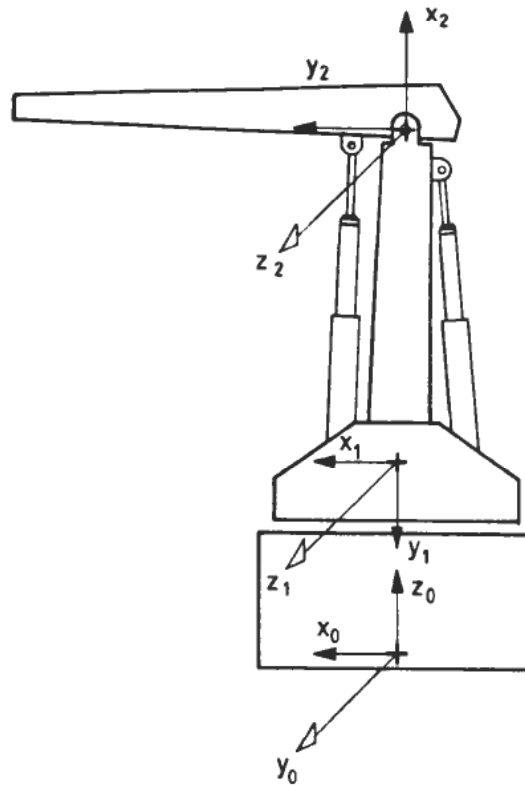


Figure 2. Manipulator used for simulation. The co-ordinate systems are assigned according to the Denavit and Hartenberg convention.

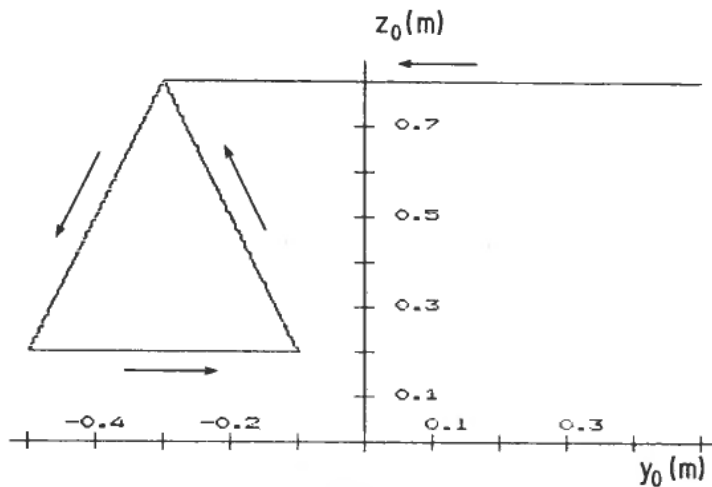


Figure 3. Position reference.

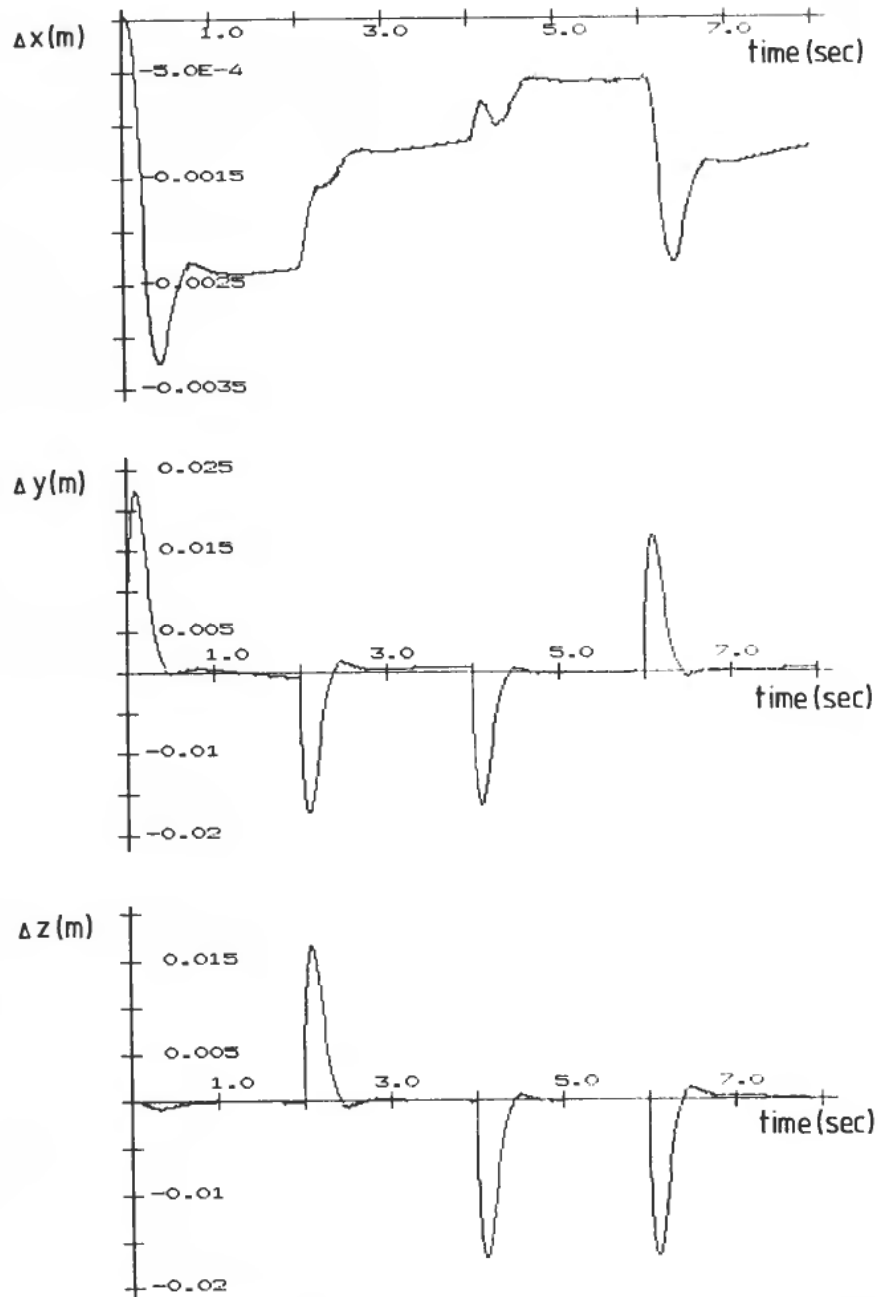


Figure 4. (a) Position deviation in the x direction. (b) Position deviation in the y direction. (c) Position deviation in the z direction.

J_{ij} are the appropriate moments of inertia and M_2 and M_3 are coefficients of gravity.

In the first simulation experiment, the end of joint three tracked the contour of a square on a workpiece. The velocity reference relative to the workpiece was 0.3 m/s. The workpiece was hanging from a conveyor which had the velocity 0.1 m/s. The resulting position reference is shown in Fig. 3. q_{ii} , $q_{i+n, i+n}$ and p_i were chosen as $q_{ii} = 10000$, $q_{i+n, i+n} = 0$ and $p_{ii} = 1$. This corresponds to choosing 1 mm as the

maximum acceptable deviation in position and 1 m/s^2 as the maximum acceptable acceleration resulting from the linear feedback control. Acceleration feedforward is not used in this experiment. In Fig. 4 the position deviations in the x , y and z directions are shown. The deviations were less than 2 mm except at the corner points where the deviations in the y and z directions were approximately 20 mm. The larger deviations at the corner points are due to the fact that infinite acceleration would be required to track the trajectory at these points. In Fig. 5 we see that

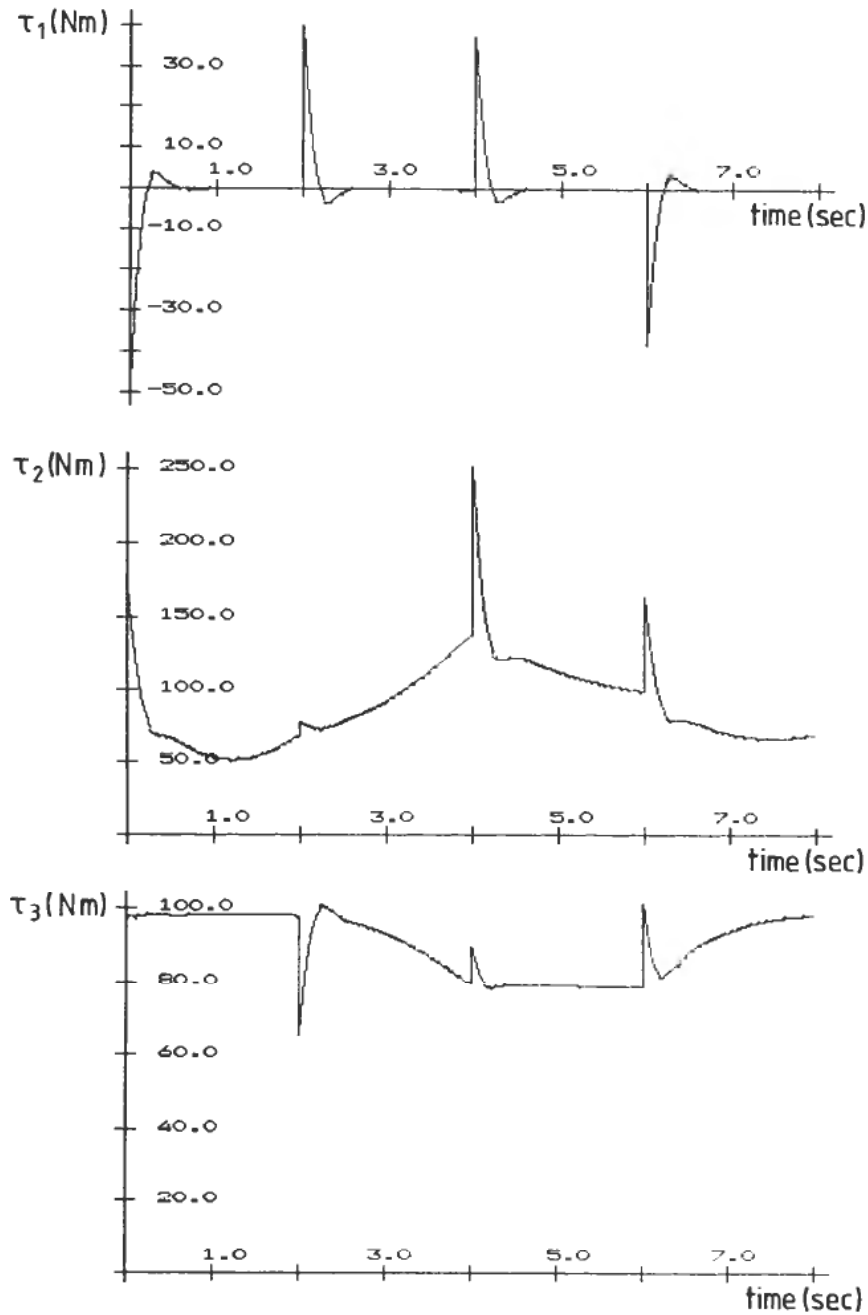


Figure 5. (a) Joint torque τ_1 . (b) Joint torque τ_2 . (c) Joint torque τ_3 .

the applied torques are well-behaved. In Fig. 6 the joint angles are shown. We clearly see that a straight line in the Cartesian space is not achieved with linear interpolation between the end-points in the joint space.

In the second simulation experiment, the manipulator lifted a mass which was unknown, 1.0 m in the z direction. Here the same control parameters were used as in the first simulation experiment. Acceleration feedforward was utilized.

The acceleration reference in the z direction was 10 m/s^2 for 0.05 s, 0 m/s^2 for

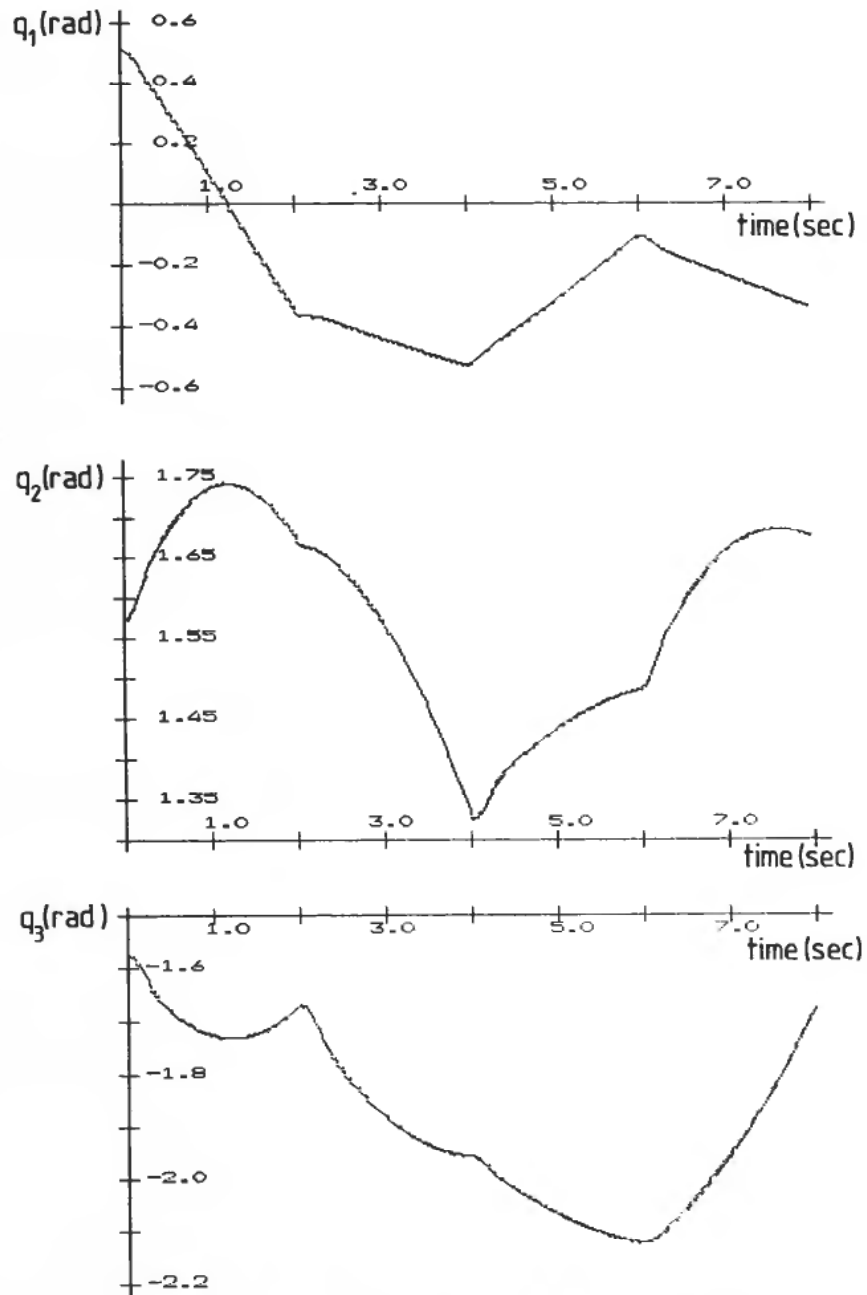


Figure 6. (a) Joint angle q_1 . (b) Joint angle q_2 . (c) Joint angle q_3 .

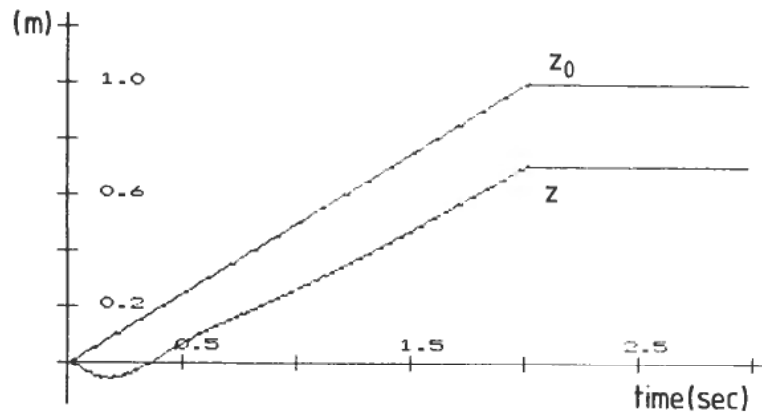


Figure 7. Position reference and position in the z direction.

1.95 s, -10 m/s^2 for 0.05 s and finally 0 m/s^2 . A mass of 20 kg was lifted. This is a rather heavy load, as links 2 and 3 weighed 25 kg and 20 kg, respectively.

In Fig. 7 the position reference in the z direction and the position of the mass is shown. Due to the unknown mass, the resulting deviation at the end-point was 300 mm. This is not acceptable. We therefore include integral action in the z direction. K_I was chosen as $K_I = 350$. The result of the new simulation with integral action is shown in Fig. 8. We see that the accuracy is very good. The applied torques τ_2 and τ_3 are shown in Fig. 9. Maximum available torque was 500 Nm. We see that τ_2 is saturated during the initial acceleration. The control is very well behaved. Instead of using integral action to compensate for the unknown load, we could have estimated the load in some way and adjusted our control. However, using integral action here is very simple, and besides it will compensate for other constant disturbances.

5. Conclusion

By means of non-linear feedback, a linear state space model describing manipulator motion in the task space is obtained. By using optimal control theory, an appropriate feedback controller for this linear system is easily found, and it is shown

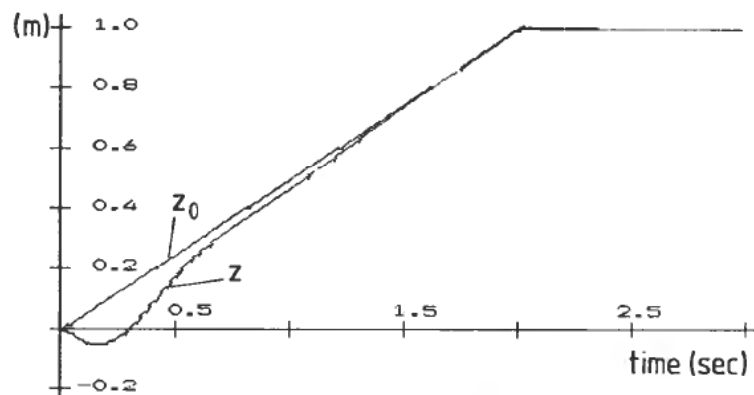


Figure 8. Position reference and position in the z direction when integral action is applied.

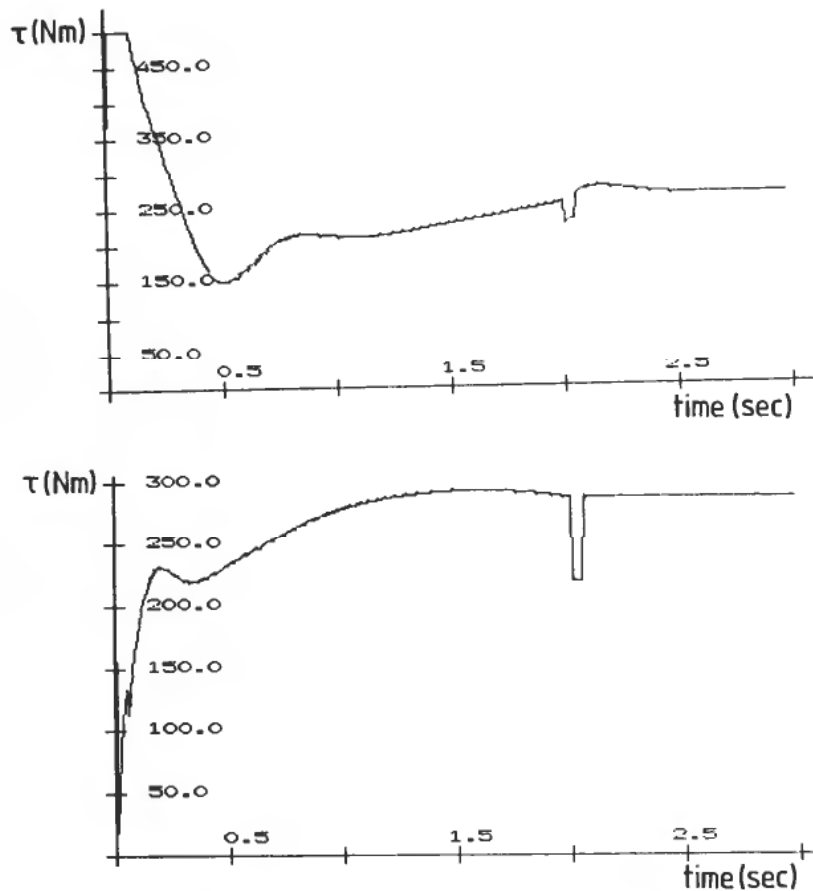


Figure 9. (a) Joint torque τ_2 . (b) Joint torque τ_3 .

by means of simulation that the resulting controller gives a good performance for task space trajectory tracking. To compensate for unmodeled effects integral action is included, and by means of simulation it has been shown that this effectively compensates for an unknown load of significant mass.

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